# **Numerical and experimental study of micro-convex dimple developed by laser additive manufacturing for surface applications**

# **Vijay Mandal\* , Vikas Tiwari, Moloy Sarkar, Sudhanshu S. Singh, J. Ramkumar**

*Indian Institute of Technology Kanpur, India*

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## **1. Introduction**

Numerous existing texturing fabrication techniques are currently being used in various applications in surface engineering. In present decade, surface texturing has become an essential tool for enhancing surface properties of the components in the field of aerospace, medical and different industrial applications made by different techniques. Lasers have become one of the most useful and accurate tools for surface engineering. This is due to lasers having large temperature gradients and cooling rate (Etsion, 2005; Mandal et al., 2022; Sharma et al., 2019).

In SLM, the different laser scanning patterns induce protuberances on the surface. The surface texture is induced by the induced surface irregularity. For instance, (Simonelli et al., 2014) scan the entire area with parallel scan direction. The scan rotation was changed by  $67^{\circ}$  for each new layer. It was reported that crystallographic texture formed in three orthogonal planes. (Dinda et al., 2012; Wei et al., 2015) scan the laser beam in forward and backward directions, observing that in unidirectional, a fibre texture is formed, while in bi-directional, a cube texture is formed. The texture on the surface is also induced by insufficient laser beam intensity, spatter formation

\*Corresponding author E-mail: vijaym@iitk.ac.in

during scanning, and a partially melted particle on the surface that is loosely associated with the surface. A large number of partially or moderately melted powder particles on the surface induces hydrophobicity with an average contact angle of 102º (Jiao et al., 2018; Sarker et al., 2018). Another way to develop texture on a 3D component is to generate a micro size feature on the surface. The surface micro-feature is intended to improve drag and surface interaction with viscous fluids. The various alternative geometry and orientation have been accomplished with fine micro features. The microfeature, i.e., track width and depth of less than 50 μm size, was achieved (Otero et al., 2012; Wang et al., 2016). Furthermore, texture surfaces were generated by varying hatch space and scanning strategy. Experiment results show that increasing the hatch space surface morphological roughness increases (Romero et al., 2011). As previously discussed, a series of articles on surface texturing using additive manufacturing techniques have been published. However, only a few numerical model articles have been presented, and only the thermal field has been studied during texturing by additive technique (Zhou et al., 2015). A 2D model was developed to examine the thermal and fluid flow dynamics in the melt pool. In this research, a thermofluidic mechanism, heating and solidification, was discussed and validated using experimental results. In addition, roles of driving forces i.e., Marangoni force, thermal buoyancy force, recoil pressure,

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and evaporation have all been investigated. In this study, the experimental results (i.e., optical profilometry images) shows good agreement with the numerical model.

#### **2. Numerical Simulation**

To understand the melt pool contour and solidification behavior, a numerical model has been established in COMSOL Multiphysics under the following assumptions:

- 1. The liquid metal flow in the melt pool is assumed to be laminar, Newtonian and incompressible.
- 2. The properties of the material are homogenous and isotropic. The thermo-physical properties of IN718 depend on temperature in solidus and liquidus phases.
- 3. The multiple reflections in the powder bed are not considered in this model.
- 4. The powder distribution in a given dimension on the top surface of the substrate material is placed manually, and powder size having 25 µm diameter.
- 5. The atmospheric air is treated as non-reactive.

#### *2.1. Governing equations*

The transport phenomena in solid, liquid, and gaseous regions are evaluated simultaneously using mass conservation, momentum conservation, and energy conservation equations [2].

$$
\frac{\partial (\rho \vec{u})}{\partial t} + \nabla \cdot (\rho \vec{u}) = 0 \qquad \qquad \ldots \qquad \ldots \qquad (1)
$$

$$
\frac{\partial (\rho \vec{u})}{\partial t} + \vec{u}.\,\nabla (\rho \vec{u}) = \nabla \cdot \left[ - p \boldsymbol{I} + \mu (\overline{\nabla \boldsymbol{u}}) + (\overline{\nabla \boldsymbol{u}})^T \right] - \rho \big( 1 - \beta (T - T_m) \big) \vec{g} + \vec{F}
$$

...................(2)

$$
\frac{\partial(\rho C_p T)}{\partial t} + \nabla .(\rho \vec{u} C_p T) = \vec{\nabla} \cdot (k \vec{\nabla} T) + \mathsf{Q} \tag{3}
$$

$$
F = -C \frac{(1 - f_l)^2}{(f_l^3 + b)} \vec{u}
$$
............(4)

$$
f_l = \begin{cases} 0 & T \leq T_s \\ \frac{T - T_s}{T_l - T_s} & T_s \leq T \leq T_l \\ 1 & T > T_l \end{cases}
$$
.................(5)

Where,  $\rho$ ,  $\overrightarrow{u}$ ,  $C_p$ ,  $\mu$ ,  $k$ ,  $\beta$ ,  $T_m$ ,  $g$  and T are listed in the table 2, and I is the identity matrix, P is pressure (Pa), F is the Darcy damping force, and Q is the net available heat for melting. Equations (4-5) evaluated from Kozeny-Carman equation, which is treated as porous media i.e., mushy zone. In equation 4, the term C is the morphology constant, which has a large value, i.e.,  $2 \times 10^7$ 

(Gan et al., 2017). Another constant term b is a small value, i.e., 10<sup>-5</sup>, in order to avoid the term  $\vec{F}$ to be infinity.

The gas liquid interface is defined by (Mandal et al., 2023)

$$
\delta = 6|\phi(1-\phi)| |\nabla \phi|
$$
.................(6)

The heat flux boundary condition with the absorption coefficient is defined by:

$$
Q = \delta * (\alpha q - h[T - T_{amb}] - \epsilon \sigma [T^4 - T_{amb}^4])
$$
 ....... (7)

$$
q = \frac{2P}{\pi r^2} \exp\left(-2\frac{x^2}{r^2}\right) \times f(t) \qquad \qquad \dots \dots \dots \dots \dots \dots (8)
$$

$$
f(t) = \begin{cases} 1, t \le t_p \\ 0, t > t_p \end{cases}
$$
.................(9)

In this equation (7,8), P, r,  $\alpha$  and δ represents the laser power, beam spot diameter, absorptivity and interface thickness respectively. q is the input heat flux,  $\epsilon$  is the emissivity and  $\sigma$  is the Boltzmann constant. The convective heat loss from the boundaries 1, 2, 3, 4, 5 and 6 are subjected to the equation:

$$
-k\nabla T = h(T - T_{amb})
$$
\n
$$
....
$$
\n(10)

At the vapor-liquid interface, the boundary condition can be defined by (Bayat et al., 2019):

$$
[-p I + \mu (\nabla \vec{u} + (\nabla \vec{u})^T)]\vec{n} = \delta * [\sigma_s \kappa \vec{n} + \nabla_S \sigma_s] \cdot (11)
$$

where,  $\mu$ ,  $\sigma_{\rm s}$  and  $\nabla_{\rm s}$  denotes the dynamic viscosity, surface tension, and gradient operator.



#### **Table 1**

Parameters used in simulation.



#### **Table 2**

Physical properties IN718 (Knapp et al., 2019; Mandal et al., 2023; Mandal et al., 2022; Sharma et al., 2019).

<b>Properties</b>	<b>Symbol</b>	Value
Solidus temperature (K)	$T_s$	1533
Vaporization temparature (K)	$T_{\rm v}$	3120
Liquidus temperature (K)	T,	1609
Density in solid (kg-m <sup>-3</sup> )	$\rho_{s}$	8193.3
Density in liquid (kg-m <sup>-3</sup> )	$\rho_{\rm I}$	7900
Temperature dependent specific heat capacity $(J-kg^{-1}-K^{-1})$	$C_{p}$	460-894.8
Temperature dependent thermal conductivity $(W-m^{-1}-K^{-1})$	k	$11.1 - 28.3$
Viscosity (Pa-s)	μ	0.00531
Thermal expansion coefficient $(K-1)$	β	$1.4 \times 10^{-5}$
Surface tension coefficient $(N-m^{-1})$	σ	1.82
Thermo-capillary coefficient (N-m <sup>-1</sup> -K <sup>-1</sup> )	$\partial \sigma / \partial T$	$-0.37 \times 10^{-3}$
Latent heat of fusion $(J-Kg^{-1})$	L	$2.27 \times 10^{5}$
Latent heat of evaporation $(J-Kg^{-1})$	$L_v$	$6.3 \times 10^6$
Absorption coefiicient	α	0.5
Atomic mass (gram)	m	58.6934
Atmosheric pressure $(N-m^{-2})$	$P_{atm}$	101300
Ideal gas constant $(J-K^{-1} - MO1^{-1})$	$R_{0}$	8.314
Convection coefficient $(W-m^{-2}-K^{-1})$	h	100

The mass flow rate of a vaporized particle is described by the Hertz-Langmuir relation as (Hirano et al., 2011):

$$
q_{\nu} = M_{\nu} \times L_{\nu} \tag{12}
$$

where,

$$
M_{\nu} = \sqrt{\frac{m}{2\pi k_b T_s}} \times P_r(T_s) \times (1 - \beta_r)
$$

where,  $m$ ,  $k_b$  and  $L_v$  represent the atomic mass, Boltzmann constant and latent heat of vaporization, respectively. The Clausius-Clapeyron relation determines the vapour pressure,  $P_r(T_s)$  as follows (Hirano et al., 2011):

$$
P_r(T_s) = 0.54 \times P_{atm} \times exp\left(\frac{M_a L_v}{R} \left(\frac{1}{T_v} - \frac{1}{T_s}\right)\right) \quad \dots (13)
$$

where,  $M_a$ , R, T<sub>v</sub> and  $\beta_r$  represent the molar mass, ideal gas constant, boiling temperature, and retro diffusion coefficient, respectively.

#### **3. Material and Parameters used for Simulation**

In this experiment, IN718 has been taken for both powder particles and substrate. Thermal properties and physical of IN718 alloys are shown in Table 2. The thermo-physical properties such as thermal conductivity, density, viscosity, and specific heat capacity are temperature dependent parameters. The processing parameters are listed in Table 1. The above model was used to simulate the laser micro-convex dimple using laser additive manufacturing with a pulse width of 0.2 ms. The average calculated laser beam diameter is 120 µm, with a laser power of 100 W and a pulse width of 0.2 ms. The process parameters were chosen based on the pilot experiment results and available reports. To better understand the dynamics of thermal and fluid flow in the melt pool, a numerical study has been included. It is important to note that the laser beam intensity emitted on the surface is in the order of 6.5\*10^9 W/m^2. Due to very high laser beam intensity, the powder bed is melted within a few microseconds.

#### **4. Results and Discussion**

## *4.1. Study of temperature field*

Fig. 2 demonstrate the temperature field during heating (0 ms to 0.2 ms) and cooling (0.2 ms to 1 ms).



**Fig. 2.** Shows the temperature field distribution during heating (a-d) and cooling (e-h). Temperature is in kelvin (K).

Fig. 2(a-d) depicts the melt flow dynamics during heating at different time period such as t= 0.096, 0.13, 0.16 and 0.162 ms. Due to increase in temperature (T<sub>max</sub> = 3350 K at 0.096 ms, T<sub>max</sub> =  $\sim$ 3390 K at 0.13 ms,  $T_{\text{max}} = \frac{6}{100}$  K at 0.16 ms and  $T_{max}$  = ~ 3400 K at 0.162 ms), recoil pressure predominant at the center of the melt pool. The melt pool is depressed at the center due to recoil pressure acting in a downward direction. The depressed liquid is  $\sim$  30 µm deep from the top surface of the powder bed (Fig. 2d).

Fig. 2(e-h), illustrate the thermo-fluidic phenomena when the laser beam is switched off. The temperature of the melt pool is  $\sim$  2900 K which is below the vaporization temperature (~ 3120 K) at 0.218 ms. The depressed melt pool starts to flow under the influence of thermocapillary forces, surface tension and gravitational force, respectively. From the Fig. 2(e-h), it has been observed that with decrease in the melt pool temperature, the depressed liquid to grew approximately  $\sim$  26 µm, which is  $\sim$  4 µm less than the actual layer thickness in the computational domain.

## *4.2. Experimental validation*

Fig. 3(a-d) represent the 3D optical profilometry images from  $1^{st}$  to  $4^{th}$  pulses. The obtained width and height of the dimples from the optical profilometry data are shown in Fig. 3. (e-h). The optical profilometry images show that as the



**Fig. 3.** (a-d) Illustrates optical profilometry images, (e-h) represent the 2D surface profile from optical profilometry data in X and Y axes, and (i-l) shows the numerical results after solidifications.

number of pulses increases, the height of the micro-convex dimple increases. A numerical simulation has been developed to validate the experimental results of the geometrical dimension of the micro-convex dimple.

Fig. 3(i-l), represents the numerical results after solidification for  $1<sup>st</sup>$ , to  $4<sup>th</sup>$  pulses respectively. The average height of experimental (i.e., optical profilometry result) and simulated results from the first to fourth pulses has been calculated. The above results show that the numerical results are very similar to the experimental results. The maximum and minimum height differences between experimental and numerical results are  $\sim$  12.5 µm and  $\sim$  1.6 µm, respectively. However, the maximum and minimum width differences of the micro-convex dimple are  $\sim$  27  $\mu$ m and  $\sim$  20  $\mu$ m, respectively. According to the findings of the preceding study, the experimental results are in good agreement with the numerical results.

## **5. Conclusions**

In this study, a 2D finite element model (FEM) was produced to understand the thermo-fluidic behavior of the melt pool, and experiments were performed to understand the surface characteristics in the micro-convex dimple. Based on this model, after reaching the vaporization temperature of IN718 ( $\sim$  3120 K) in the melt pool, recoil pressure dominates, resulting in a depression in the melt pool. When the laser is turned off, the temperature in the melt pool begins to fall. As the temperature in the melt pool falls below the vaporization temperature, the depressed melt pool begins to rise due to the influence of thermocapillary forces. The maximum and minimum differences in the melt pool height between experimental and numerical values are  $\sim$  12.5  $\mu$ m and  $\sim$  1.6 µm, and the melt pool width are  $\sim$  27 µm and  $\sim$  20 µm, respectively.

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**Dr. Vijay Mandal** is a Post Doctoral Fellow in the department of Metallurgical and Materials Science at Indian Institute of Technology Bombay. He obtained his Ph.D in Mechanical Engineering from IIT Kanpur in 2022. His

research focuses on the laser powder bed fusion (L-PBF) additive manufacturing process.



**Vikas Tiwari** is a Research Scholar in the Department of Materials Science and Engineering at Indian Institute of Technology Kanpur. He obtained his B.Tech degree from Maulana Azad National

Institute of Technolog Bhopal. (E-mail: vikast20@iitk.ac.in)



**Moloy Sarkar** is a Research Scholar in the Department of Mechanical Engineering at Indian Institute of Technology Kanpur. He obtained his B.Tech degree from Jalpaiguri Government

Engineering College, West Bengal. (E-mail: moloys21@iitk.ac.in)



**Dr. Sudhanshu Shekhar Singh**  is an Associate Professor in the Department of Materials Science and Engineering at Indian Institute of Technology Kanpur. He obtained his Ph.D degree

from Arizona State University. He is a recipient of NASI-Young Scientist Platinum Jubilee Awards 2020 by The National Academy of Sciences (India), 2018SMD JOM Best Paper Award by The Minerals, Metals & Materials Society (TMS), Young Metallurgist of the Year Award 2017 by Ministry of Steel & Mines (India) and IEI Young Engineers Award 2017–2018 by The Institution of Engineers (India). (E-mail: sudhanss@iitk.ac.in)



**Dr. J. Ramkumar**, Professor in the Department of Mechanical Engineering and Design Program at Indian Institute of Technology Kanpur, has over 20 years of experience in industry, research

and academia. He obtained his Ph.D in Mechanical Engineering from Indian Institute of Technology Madras. He joined as an Assistant Professor in the Department of Mechanical Engineering in Dec 2003. He has published over 330 articles in the peer-reviewed international journals and has delivered over 100 lectures in the international conferences. He has 4000+ citations for his publications & his h-index of 33, which endorses his high research productivity. Five of his patents are commercialized, total number of patents in his name being 88. Currently, he is the Project Investigator of MedTech facility, Imagineering Laboratory and RuTAG facility at IIT Kanpur. (E-mail: jrkumar@iitk.ac.in)